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Finite Element Analysis of a Friction Clutch System in an Automatic Transmission

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Finite Element Analysis of a Friction Clutch System in an Automatic Transmission

CAPSTONE DESIGN EXPO 2017

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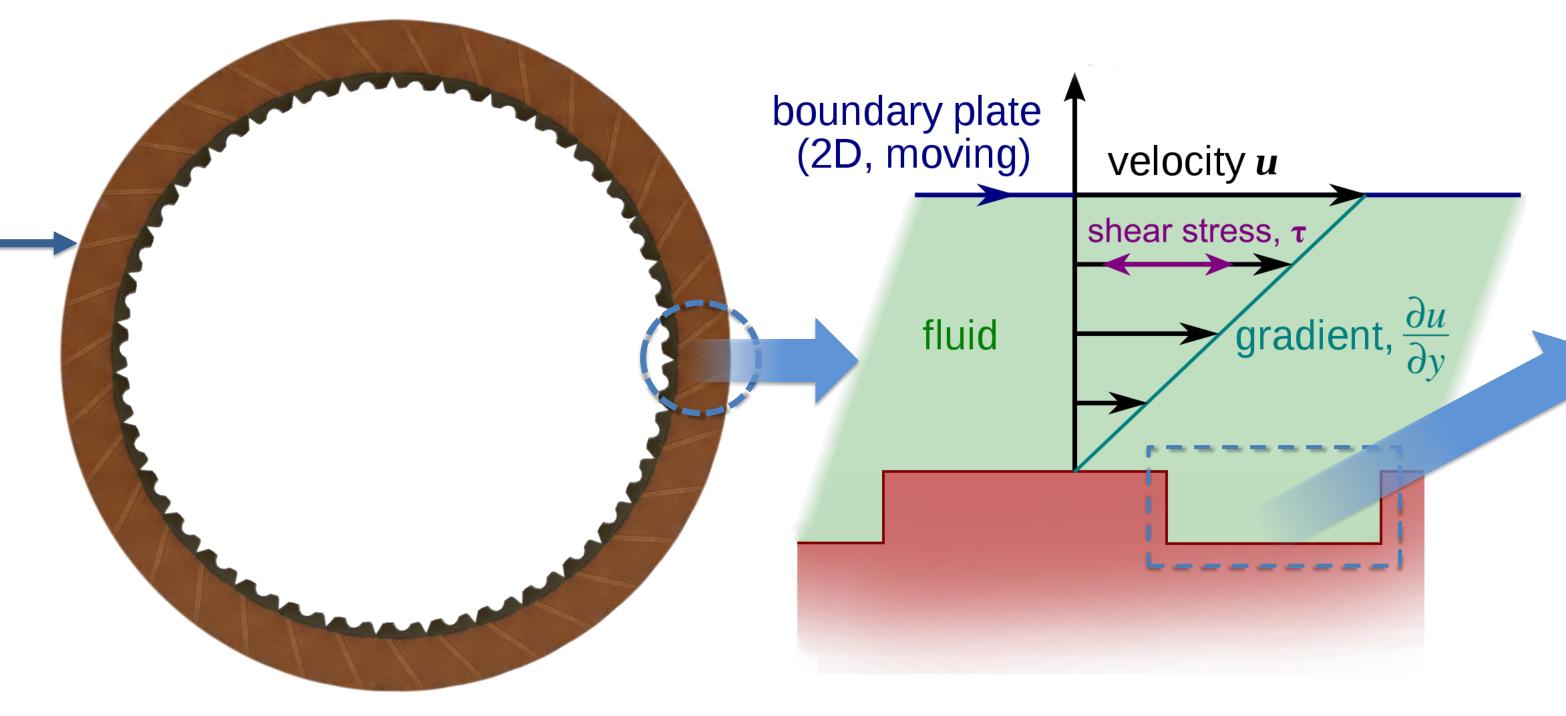
Objective

Develop a computational solution to test parameters that affect torque fluctuations in a friction clutch system



Solution

The solution is a two dimensional FEA simulation that is cost effective and time efficient.



Operating Conditions & Huid Parameters					Friction Material Properties and Plate Geometry				
Category	Symbol	Value	Units	Description		Ε	30	MPa	Young's elastic modulus
0				-	Friction Material	σ	6	μm	Surface roughness
Operating Conditions	P _{app}	1000	kPa	Plate-specific apply pressure		Da		_	Asperity density
	h₀/σ	4	-	Initial film thickness ratio		β	0.25	mm	Asperity tip radius
	$\omega_{initial}$	3000	rpm	Initial Slip Speed		\mathbf{A}_{Grv}	5	%	Percent groove area
	initial					d	0.5652	mm	Thickness
	ω_{final}	0	rpm	Final Slip Speed		Kper	1.00E-13	m^2	Permeability
	T _{Hnitial}	80	℃	Initial Temperature	Plate Geometry	n	6	_	Number of friction surfaces
	T_{final}	250	°C	Final Temperature		r,	46.83	mm	Inner radius of friction lining
			%	Percentage of area supporting Papp		r _o	58.74	mm	Outer radius of friction lining
	A _{Load}				Derived Parameters				
	t _{end}	4	s	Time for full engagement		Rc	53.0	mm	Effective clutch radius
Fluid Properties	λ	4.99E-03	Pa∙s	Viscosity (at P-v-T conditions)	Plate Geometry	A_N	3950.0	mm ²	Friction interface area
	α	5	1/GPa	Pressure-Viscosity coefficient		A _{net}	3752.5	mm ²	Friction area net of grooves
	μ_{a}	0.14	_	Coefficient of asperity friction	Afton® Passion for Solutions™				
	dμ _a /dT	-4.0E-04	1/°C	Temperature gradient of μ_{a}					
	ε	30	nm	Minimum film thickness					

Results

The simulation is simple to set up, takes very little time to generate values, and yields accurate results based on prior knowledge in fluid mechanics.

